

**3E1413**

Roll No. : \_\_\_\_\_

Total Printed Pages : **3**

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**B. Tech. (Sem.III) (Main/Back) Examination, January - 2009**  
**(3ME3) Engineering Thermodynamics (Mechanical Engg.)**  
**(3PI3) Engineering Thermodynamics (Prod. & Indus. Engg.)**  
**(3AE3) Engineering Thermodynamics (Automobile Engg.)**

Time : **3 Hours**]

[Total Marks : **80**

[Min. Passing Marks : **24**

*Attempt five questions in all. Schematic diagrams must be shown wherever necessary. Any data you feel missing may suitably be assumed and stated clearly.*

Use of following supporting material is permitted during examination.  
(Mentioned in form No. 205)

1. STEAM TABLE

2. MOLLIER CHART

1 (a) Define the following thermodynamic terms with suitable examples :

- (i) System
- (ii) Properties
- (iii) Process
- (iv) Cycle.

**2+2+2+2**

(b) What is temperature scale ? How is temperature scale established ? Explain standard fixed point in thermometry.

**3+3+2**

**OR**

1 What is a pure substance ? Draw phase equilibrium diagram of water on P-V coordinates and define the different saturation states, critical point and triple point.

**2+4+6+2+2**

2 Air enters a heat exchanger at 15°C temperature, 30 m/s velocity and is heated to 800°C. It then enters a turbine with 30 m/s velocity and expands until the temperature falls to 650°C. On

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1

[Contd...

leaving the turbine, the air is taken at a velocity of 60 m/s to a nozzle where it expands until the temperature has fallen to 500°C. If mass flow rate is 2 kg/s, Calculate :

- (a) Heat transfer rate in heat exchanger
- (b) Power developed by the turbine
- (c) Exit velocity from the nozzle.

Assume  $C_p = 1.005 \text{ kJ/kg-K}$

5+5+6

OR

- 2 (a) Prove that maximum work obtainable from two finite heat reservoirs is given by

$$W_{\max} = C_p (\sqrt{T_1} - \sqrt{T_2})^2$$

where  $C_p \rightarrow$  Heat capacities of reservoirs

$T_1 \rightarrow$  Higher temperature

$T_2 \rightarrow$  Lower temperature.

8

- (b) Determine entropy change of the universe, when 1 kg of ice at -5°C is exposed to the atmosphere which is at 20°C.

Assume  $C_{P_{ice}} = 2.093 \text{ kJ/kg-K}$ ,  $C_{P_{water}} = 4.187 \frac{\text{kJ}}{\text{kg-K}}$

and latent heat of fusion is 333.3 kJ/kg.

8

- 3 What is available and unavailable energy ? Show that there is decrease in available energy when heat is transferred through a finite temperature difference. Explain the concept of quality of energy.

2+2+8+4

OR

- 3 (a) Derive first and second Tds equation.

8





(b) Prove that  $C_p - C_v = -T \left( \frac{\partial V}{\partial T} \right)_p^2 \left( \frac{\partial P}{\partial V} \right)_T$

What are important facts which can be deduced from the expression ?

6+2

- 4 In an air standard diesel cycle, the compression ratio is '16' and at the beginning of isentropic compression, the temperature is 15°C and the pressure is 0.1 MPa. Heat is added until the temperature at the end of the constant pressure process is 1480°C.

Calculate :

- (a) Cut off ratio
- (b) Heat supplied per kg of air
- (c) Cycle efficiency
- (d) Mean effective pressure.

4×4

OR

- 4 (a) Draw Atkinson cycle on P-V and T-S diagram and derive an expression for cycle efficiency.

3+7

- (b) Explain the working of two stroke petrol engine.

6

- 5 Steam at 0.8 MPa, 250°C and with 1 kg/s flow rate, mixes with another stream of steam at 0.8 MPa, 0.95 dry. Total flow rate is 2.3 kg/s, calculate final temperature and entropy of the steam. If the steam is now expanded in a nozzle isentropically to a pressure of 0.4 MPa, determine the final velocity at exit.

8+8

OR

- 5 In a single heater regenerative cycle, steam inlet to turbine is 30 bar, 400°C. Exhaust pressure is 0.10 bar. Feed water heater works at 5 bar. Find the cycle efficiency, steam rate and increase in mean temperature of heat addition.

5+5+6



Prove that  $\frac{dP}{P} = \frac{dV}{V} \gamma$  for an adiabatic process.

What are important facts which can be deduced from the expression  $P \gamma = \text{constant}$ ?

$$P_1 V_1^\gamma = P_2 V_2^\gamma$$

In an air standard diesel cycle, the compression ratio is 18 and in the beginning of isentropic compression, the temperature is 50°C and the pressure is 0.1 MPa. Heat is added until the temperature of the gas at the end of expansion is 1800°C.

- (a) Calculate the cycle efficiency.
- (b) Heat rejected per kg of air.
- (c) Cut off ratio.
- (d) Mean effective pressure.

Draw Airson cycle on P-V diagram and derive an expression for cycle efficiency.

Explain the working of two stroke engine on P-V diagram.

Steam at 8 MPa, 500°C and with 1 kg mass is expanded isentropically to 0.1 MPa. Calculate final temperature and volume. If the steam is now expanded in a nozzle isentropically to a pressure of 0.1 MPa determine the final velocity of exit.

OR

In a single heat engine cycle, steam enters at 10 bar, 500°C. Exhaust pressure is 0.1 bar. Heat water heater is 2 bar. Find the cycle efficiency, steam rate and increase in mean temperature of heat addition.